

TABLE LVIII. DISPLACEMENTS AND MAXIMUM HOOP STRESSES IN PIN SEGMENTS, $\nu = 0.3$

k_2	$\frac{\sigma_{\theta/p_1}}{E\nu}$ at $\theta = \alpha/4$, $r = r_2$		$\frac{E\nu}{r p_1}$ at $\theta = 0$, $r = r_1$		$\frac{E\nu}{r p_1}$ at $\theta = \alpha/2$, $r = r_1$	
	$r = r_2$	$r = r_1$	$r = r_2$	$r = r_1$	$r = r_2$	$r = r_1$
2.0	4.3266	1.0074	-0.0151	-0.6387	0.5367	0.3202
3.0	2.7247	1.0681	-0.1303	-0.5313	0.3202	0.2459
4.0	2.0126	1.1739	-0.1456	-0.5149	0.2554	0.2554
5.0	1.6019	1.2865	-0.1397	-0.4068		
(b) $k_2 = 3.0$						
2.0	3.3815	1.0516	-0.1281	-0.4082	0.2336	0.3202
3.0	2.7247	1.0681	-0.1303	-0.5313	0.3202	0.3202
4.0	2.0820	1.1137	-0.1305	-0.7382	0.5195	0.5195
5.0						
(a) $\alpha = 60^\circ$						

where A is the area of the pin and $P/2$ is the shear force shown in Figure 81. For

$A = \frac{\pi d^2}{4}$ (d is pin diameter) and P given by Equation (101), the maximum shear stress becomes

$$\tau_{\max} = \frac{16}{3} \frac{P_1 r_1 t}{\pi d^2} \quad (109)$$

This equation is the basis of Equation (69) in the text.